

What are the recommended practices in debugging a model?

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# General Approach to Modeling

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## Crawl-walk-run

- Try to understand the mechanism from a physical standpoint.
- Use building blocks of concepts that have worked in the past.
- Add enhancements to the model while testing periodically.
- Build kinematic models before building dynamic models.
- Use motions to check models before applying forces.
- Use motions which start with zero velocity.
- Verify enhancements to a complex model on a simpler model first.

# Modeling Practices: Parts

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## Geometry associativity errors

Geometry may be added to the wrong part.

## Mass properties

- Using imported CAD-created geometry (IGES, STL, and so on) can yield inaccurate mass properties.
- Ensure inertia matrix is realistic.
- Use aggregate mass for a quick check of system mass and inertia.
- Use the Table Editor to do a quick check and potentially fix individual part masses and inertia.
- Small part mass and inertia lead to unrealistically high frequencies.

## Initial velocities

Check to see that part initial velocities are consistent (look in the .out file).

## Dummy parts

- Whenever possible, avoid using them.
- If absolutely needed, constrain all six DOF and assign a mass of 0.0 (not 1e-20).

## Design configuration

- Build a model close to assembled position.
- Build a model close to a stable equilibrium position, if possible.

## Fixed joints

- Not needed, since two or more parts can be combined or merged into a single part.
- An extra part with a fixed joint adds unnecessary equations to your system.
- When locking a part to ground, enormous torque may develop due to large moment arms.
- If absolutely needed, then add fixed joints at the center-of-mass (cm) location of lightest part.
- If locking a part to ground, consider assigning a very large mass/inertia to it so it can behave like ground.

**Note:** Whenever possible, avoid using fixed joints.

## Universal joints

When a universal joint is at  $90^\circ$ , you get a singular matrix.

## Motion

- Motion elements should only be functions of time.
- Do not use motion (functions) as a function of variables (or states).

**Note:** Avoid redundant constraints.

## Spring-dampers

- Ensure that the marker endpoints ( $DM(I,J)$ ) are never superimposed.
- Watch out for springs with very stiff spring constants.
- Watch out for springs with no damping.

## Bushings

Watch out for bushings with large rotations.

# Modeling Practices: Run-time Functions

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## Function Builder

- Assists in building functions.
- Assists in function verification.
- Has function plot capability.

## Velocity

Make sure velocities are correct in force expressions. For example, in the damping function:  $-c*VX(i, j, j, \_)$ , the fourth term is missing.

## Splines

- Approximate forces with smooth, continuous splines.
- Extend the range of spline data beyond the range of need.
- Cubic splines (CUBSPL) work better on motions than Akima.
- Akima splines (AKISPL) work better on forces than Cubic.
- The Akima interpolation method is faster and can be defined as a surface, but its derivatives are generally discontinuous.

## IMPACTs/BISTOPs

- Do not use 1.0 for exponent on IMPACT or BISTOP functions.
- Models with IMPACTs/BISTOPs should have slight penetration in design position when doing statics.

## Measures

- Set up measures of your run-time functions.
- Set up measures of components of your run-time functions.
- Ensure that your function will not try to divide by zero.

# Modeling Practices: Run-time Functions

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## Contacts

- Do not use 1.0 for exponent on IMPACT or BISTOP functions.
- Models with contacts should have slight penetration in design position when doing statics.

## Tires

- Models with tires should have slight penetration in model position when doing statics.
- If only rear tires penetrate, the static position could be a “handstand.”

## Units

- Use consistent units throughout the model (time, mass, stiffness, damping, and so on).
- Choose units (mass, force, time, and so on) that do not require using very large or very small numbers.
- Be wary when your model contains numbers like  $1e+23$  or  $1e-20$ .
- Use appropriate units—when modeling large models such as an aircraft landing on a runway, length units of millimeters may not be appropriate. Conversely, when modeling small models such as a power window switch (made up of small moving parts), using length units of meters may not be appropriate.
- Use reasonable time units—high frequencies may be better modeled with time units of milliseconds rather than seconds.

## Gravity

- Check magnitude and direction.
- Check for multiple gravity elements.

## Model verify

- Lists number of moving parts, number of each type of constraint.
- Lists Gruebler's count and actual DOF count.
- Lists redundant constraints.
- Reports misaligned forces/force elements, joints, and so on.
- Helps identify and eliminate causes for input warning (don't ignore).

## Model topology

- Text or graphical model topology.
- Table Editor provides spreadsheet-like overview of model content.

## Icon feedback

Broken icon in design configuration probably means incorrectly defined joint or force.

## Table Editor

Convenient way to inspect and modify models (particularly large ones).

## Interactive simulation

By default, is turned on.



## Model display update

As ADAMS performs the simulation, you have the option to get immediate graphical feedback of the simulation at every:

- Output step
- Integration step
- Iteration

## Icons visible during simulation

This may help you monitor behavior of model components.

## Subroutines

- Check for their existence.
- While debugging a model, eliminate user subroutines so that they are not the source of the error.

## Gravity

Turning gravity off can accentuate modeling errors and make debugging easier.

## Statics

- When applicable, perform an initial static first.
- If static solution fails:
  - ◆ Turn on **Model display update = at every iteration** to provide additional insight.
  - ◆ Identify and eliminate the undesired static configuration—there could be more than one static configuration and ADAMS could be finding the undesired one.
- Check to see if there are any floating parts.
- Check the signs of applied forces.
- Experiment with Alimit/Tlimit/Maxit/Stability.
- Check if impacts are initially in contact; if not, they should be.
- Running an initial dynamic simulation can help you determine why the model is not finding static equilibrium.

## Dynamics

- If integrator fails to **start-up**:
  - ◆ Check sign and magnitude of forces.
  - ◆ Look at accelerations to understand what is happening.
  - ◆ Perform initial static analysis first.
  - ◆ Try a quasi-static simulation.
  - ◆ Try changing integrator parameter - HINIT.
  - ◆ Try a different integrator.
- If integrator fails in the **middle of a simulation**:
  - ◆ Look at animation and plots until failure, to understand simulation.
  - ◆ Decrease integrator parameter - HMAX.
  - ◆ Do not let the integrator step over important events.
  - ◆ Short duration events, such as an impulse can be captured by setting the maximum time step, HMAX, to a value less than the impulse width.
  - ◆ Use HMAX so ADAMS/Solver acts as a fixed-step integrator
  - ◆ Decrease error.
  - ◆ Try a different integrator.
- If integrator takes **very small steps**:
  - ◆ Look for sudden changes in force and motion input.
  - ◆ Rescale model to get more uniform numbers.

## Velocities at time=0

Check initial velocities using the .out file.